

ASME B29.22-2001
(Revision of ASME B29.22M-1995)

DROP FORGED RIVETLESS CHAINS, SPROCKET TEETH DRIVE CHAIN/DRIVE DOGS

AN AMERICAN NATIONAL STANDARD



The American Society of
Mechanical Engineers

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CORRESPONDENCE WITH B29 COMMITTEE

General. ASME standards are developed and maintained with the intent to represent the consensus of concerned interests. As such, users of this Standard may interact with the Committee by requesting interpretations, proposing revisions, and attending Committee meetings. Correspondence should be addressed to:

Secretary, B29 Main Committee
The American Society of Mechanical Engineers
Three Park Avenue
New York, NY 10016-5990

Proposing Revisions. Revisions are made periodically to the Standard to incorporate changes that appear necessary or desirable, as demonstrated by the experience gained from the application of the Standard. Approved revisions will be published periodically.

The Committee welcomes proposals for revisions to this Standard. Such proposals should be as specific as possible, citing the paragraph number(s), the proposed wording, and a detailed description of the reasons for the proposal, including any pertinent documentation.

Interpretations. Upon request, the B29 Committee will render an interpretation of any requirement of the Standard. Interpretations can only be rendered in response to a written request sent to the Secretary of the B29 Main Committee.

The request for interpretation should be clear and unambiguous. It is further recommended that the inquirer submit his/her request in the following format:

Subject: Cite the applicable paragraph number(s) and the topic of the inquiry.
Edition: Cite the applicable edition of the Standard for which the interpretation is being requested.
Question: Phrase the question as a request for an interpretation of a specific requirement suitable for general understanding and use, not as a request for an approval of a proprietary design or situation. The inquirer may also include any plans or drawings, which are necessary to explain the question; however, they should not contain proprietary names or information.

Requests that are not in this format will be rewritten in this format by the Committee prior to being answered, which may inadvertently change the intent of the original request.

ASME procedures provide for reconsideration of any interpretation when or if additional information that might affect an interpretation is available. Further, persons aggrieved by an interpretation may appeal to the cognizant ASME Committee or Subcommittee. ASME does not “approve,” “certify,” “rate,” or “endorse” any item, construction, proprietary device, or activity.

Attending Committee Meetings. The B29 Main Committee regularly holds meetings, which are open to the public. Persons wishing to attend any meeting should contact the Secretary of the B29 Main Committee.

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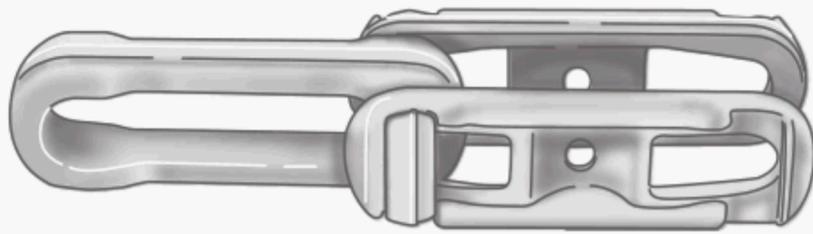


FIG. 3 X-TYPE DROP FORGED RIVETLESS CHAIN

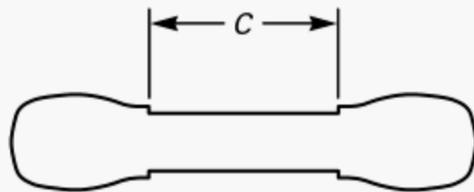


FIG. 4 X-TYPE CHAIN

4 MEASURING LOAD

The measuring load is determined as five times the weight of a 10 ft strand rounded to the nearest 100 lb. This value is then converted to kilonewtons for the metric equivalent.

5 STRAND LENGTH TOLERANCE

The length of new chains subjected to the specified measuring load must fall within the plus and minus tolerances shown in Table 1. Specific maximum and minimum strand lengths are shown in Tables 2, 3, and 4 for each chain.

Maximum and minimum strand tolerances for the given number of pitches in a measuring strand are shown in Table 1.

6 DIMENSIONS FOR CHAIN LINKS

To assure interchangeability of links as produced by different makers of chains, standard maximum and minimum dimensions are adopted. They are not actual dimensions used in manufacturing, but limiting dimensions, maximum or minimum, required to assure the desired interchangeability.

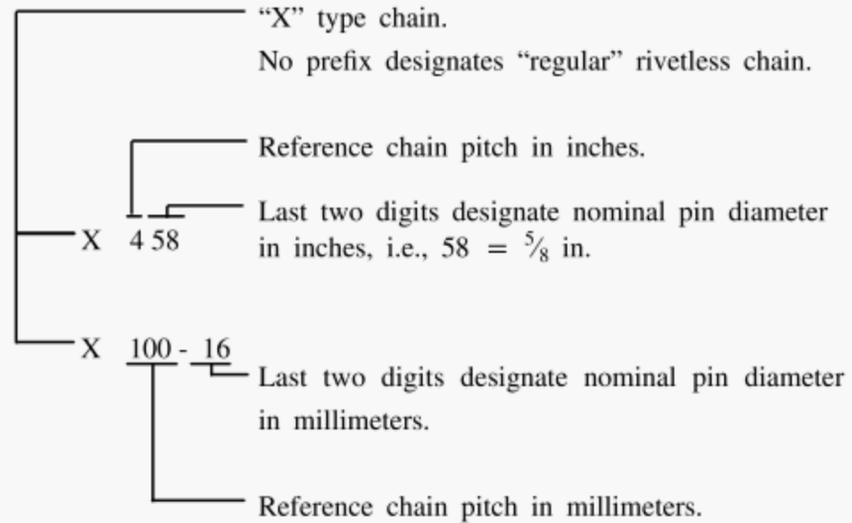
The metric equivalent dimensions are for reference only.

7 FINISH

Sharp edges and protrusions shall be absent from the pin seating and driving face areas of the center link.

8 NUMBERING SYSTEM

The numbering system for the rivetless chain gives significance to the chain number as shown.



NOTE: Modified X-type chain is identified as X-type without additional notation on the chain.

9 CHAIN ASSEMBLY

See Fig. 5 and Tables 2, 3, and 4.

10 SPROCKETS

Two sprocket tooth forms are included: Forms A and B. The major difference between these two forms is in the method of generating the tooth form. Form A produces a curved tooth face and Form B a straight tooth face. Either form may be used. See Tables 5, 6, and 7, and Figs. 6 and 7.

10.1 Sprocket Tooth Form Data

Root diameter (D_r), pocket radius (R_p), and outside diameter (D_o) must not exceed the values obtained by the formulae. Oversize dimensions may cause improper chain and sprocket interaction and excessive chain loads (see Fig. 6).

In some cases the outside diameter (D_o) may be limited by special attachments mounted on the chain. For this reason, the outside diameter (D_o) should be checked to assure that interference does not exist. D_o obtained by the formula should be rounded off to the next smallest $\frac{1}{8}$ in. (3 mm).

Chain clearance circle (C_c) established by the formula is the maximum and will provide clearance under most conditions. However, the value obtained by the formula should be checked to assure that a given hub diameter does not interfere with chain attachments.

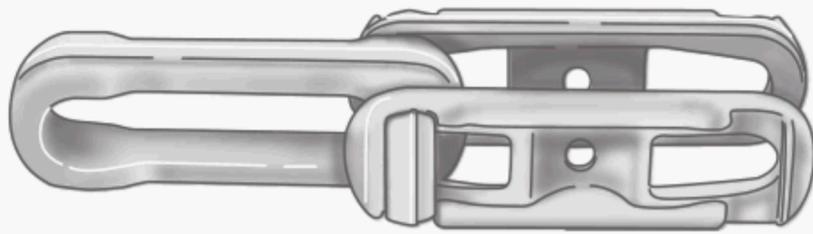


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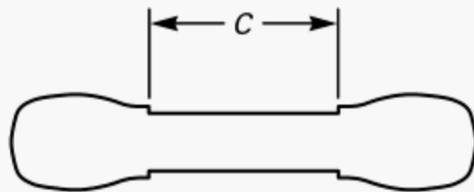


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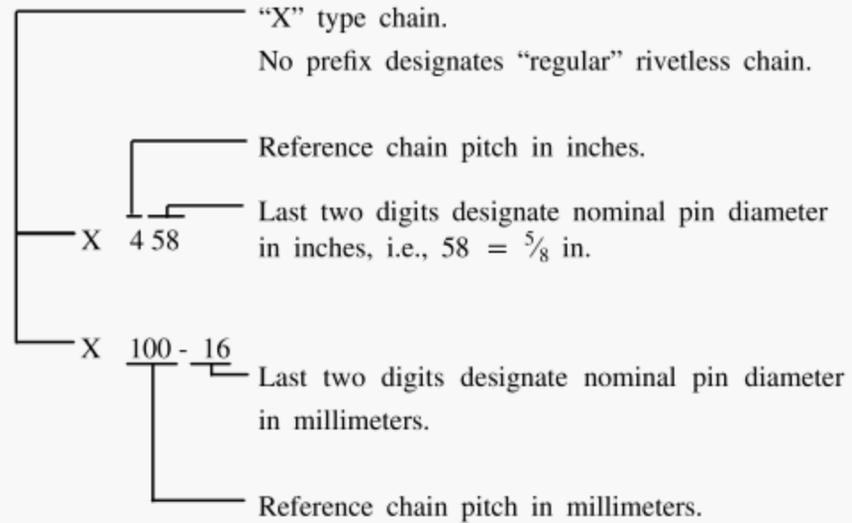
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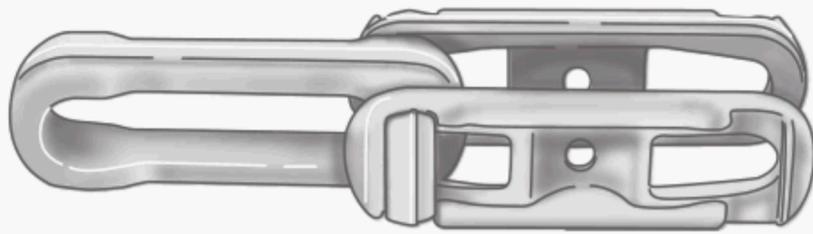


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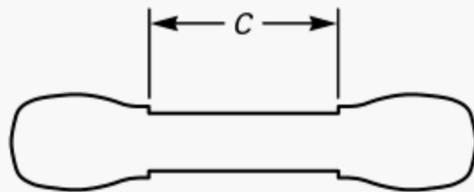


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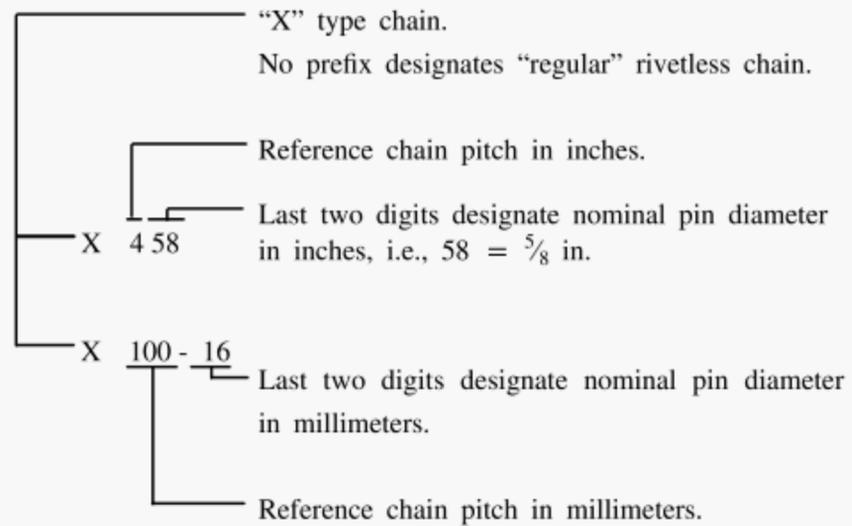
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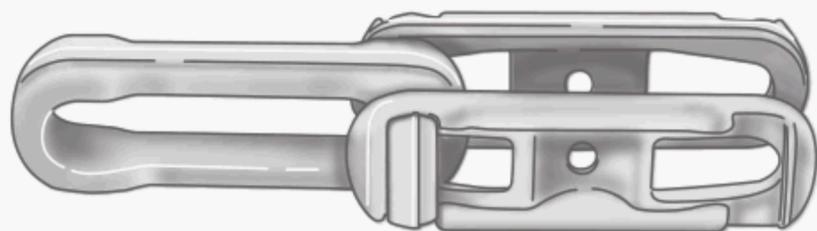


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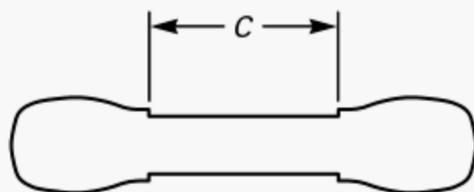


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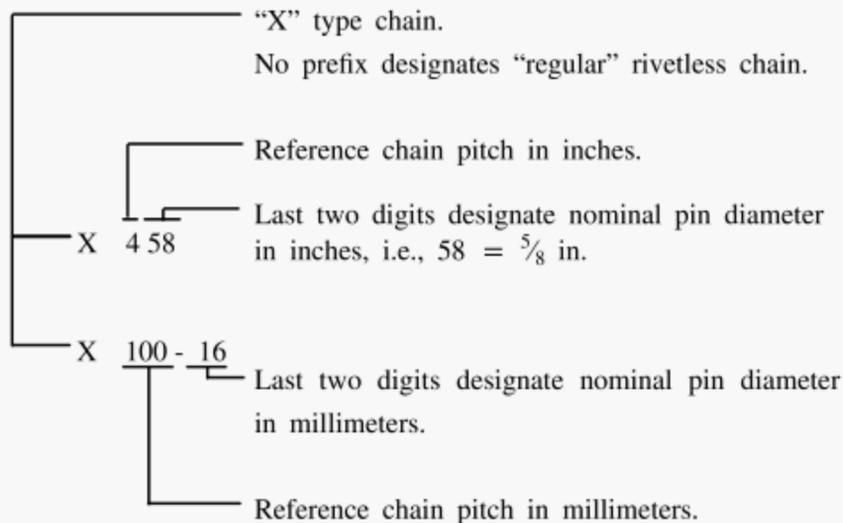
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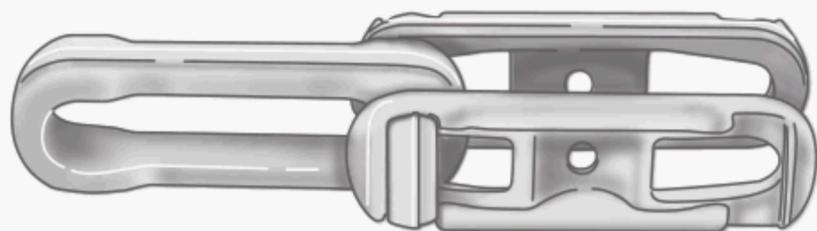


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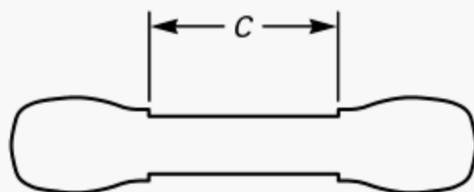


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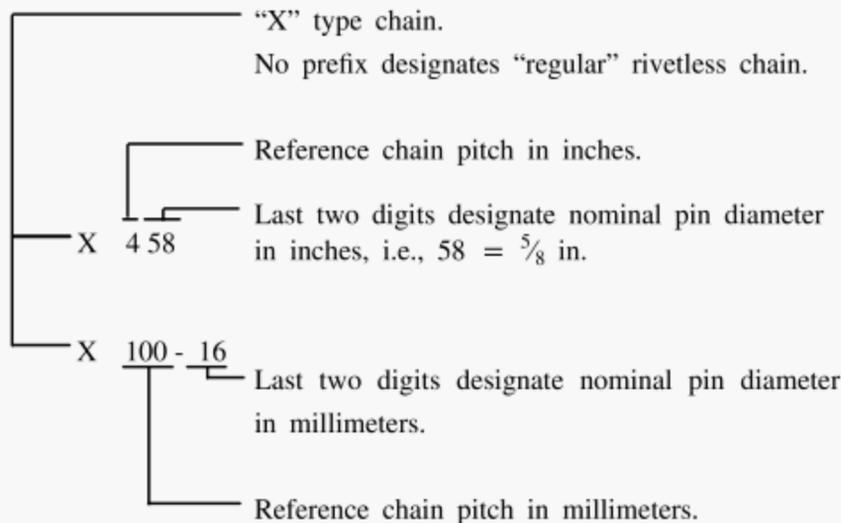
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Chain clearance circle (C_c) established by the formula is the maximum and will provide clearance under most conditions. However, the value obtained by the formula should be checked to assure that a given hub diameter does not interfere with chain attachments.

**TABLE 4 MODIFIED X-TYPE DROP FORGED RIVETLESS CHAIN
General Chain Dimensions, Minimum Ultimate Strength Rating, Strand Length, and Measuring
Load for Checking Chain Lengths**

Chain No.	Dimensions, in. (mm)					
	X348 (X75-13)		X458 (X100-16)		X678 (X150-22)	
	in.	(mm)	in.	(mm)	in.	(mm)
Reference pitch	3	(75)	4	(100)	6	(150)
<i>P</i>	3.015	(76.6)	4.031	(102.4)	6.031	(153.2)
<i>B</i> - Min.	0.53	(13.5)	0.66	(16.8)	0.95	(24.1)
<i>C</i> - Min.	1.59	(40.4)	2.31	(58.7)	3.34	(84.8)
<i>D</i>	0.49	(12.4)	0.63	(16.0)	0.87	(22.1)
<i>F</i> - Max.	1.10	(27.9)	1.44	(36.6)	2.03	(51.6)
<i>L</i> - Max.	1.73	(43.9)	2.25	(57.2)	3.03	(77.0)
<i>T</i>	0.40	(10.2)	0.48	(12.2)	0.70	(17.8)
<i>X</i>	0.74	(18.8)	1.00	(25.4)	1.28	(32.5)
<i>X_a</i>	0.51	(13.0)	0.64	(16.3)	0.83	(21.1)
<i>Z</i>	0.79	(20.1)	1.07	(27.2)	1.35	(34.3)
Minimum ultimate tensile strength, lb (kN)	22,000	(98)	42,000	(187)	72,000	(320)
No. of chain pitches in standard measuring length	40	(40)	30	(30)	20	(20)
Standard measuring length (in.)						
Max.	121.85	(3095.0)	121.68	(3090.7)	121.37	(3082.8)
Min.	120.10	(3050.5)	120.43	(3058.9)	120.37	(3057.4)
Measuring load, lb (kN)	100	(0.4)	200	(0.9)	300	(1.3)

CAUTION: The numerical values in this table must be read in conjunction with the definition and explanatory note appearing in Section 6, "Dimensions for Chain Links." The M.U.T.S. values do not provide a sufficient or appropriate basis for determining chain application.

**TABLE 5 SPROCKETS:
Maximum Eccentricity and Face Runout at Root Diameter**

Pitch Diameter				Max. Face Runout TIR		Max. Eccentricity TIR	
Over, in.	Including, in.	Over, mm	Including, mm	in.	mm	in.	mm
0	up to 12	0	up to 305	0.06	1.52	0.09	2.29
12	up to 24	305	up to 610	0.12	3.05	0.15	3.81
24	up to 36	610	up to 915	0.20	5.08	0.21	5.33
36	up to 48	915	up to 1220	0.30	7.62	0.27	6.86
48	up to 60	1220	up to 1524	0.33	8.38	0.33	8.38
60	up to 72	1524	up to 1830	0.36	9.14	0.39	9.91

GENERAL NOTE: For pitch diameter over 72 in. (1830 mm), consult manufacturer.

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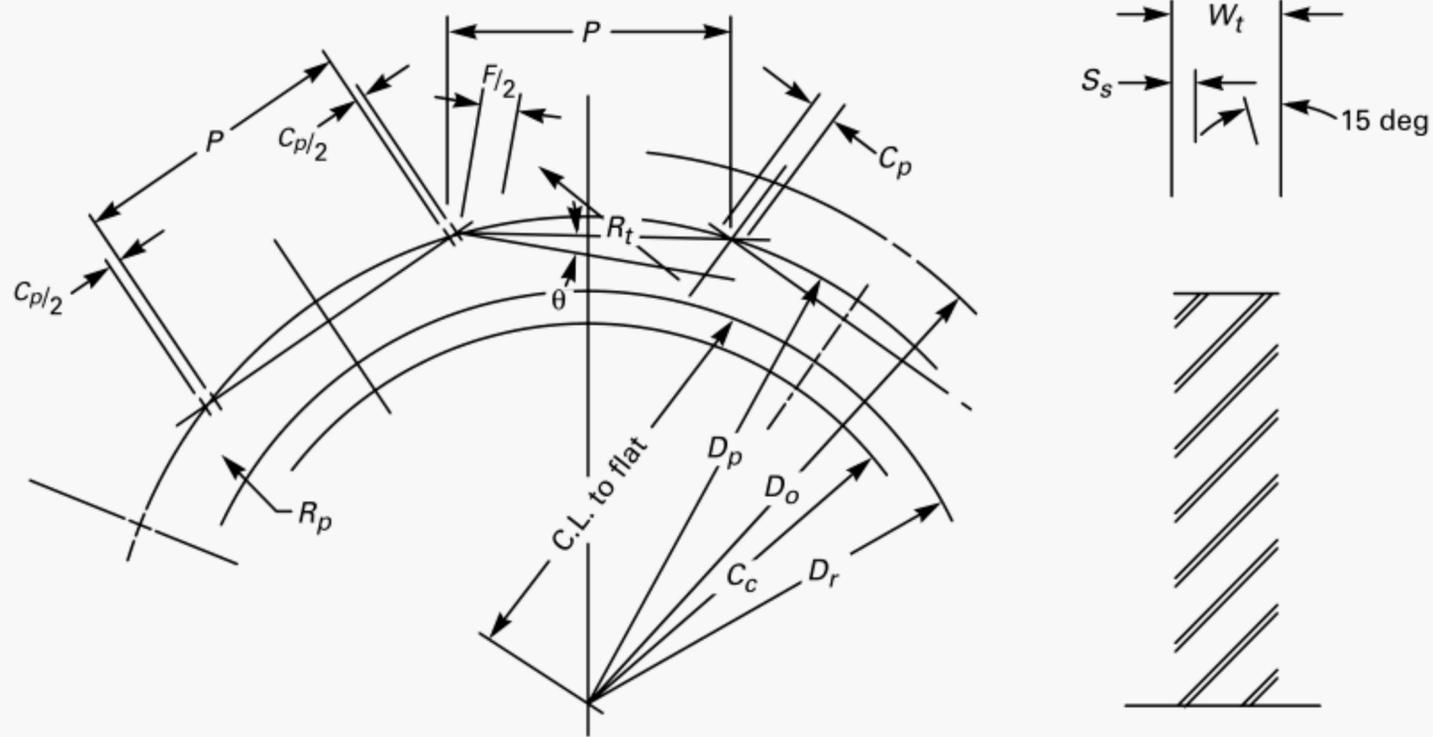
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GENERAL NOTE: For pitch diameter over 72 in. (1830 mm), consult manufacturer.



The elements of sprocket tooth form may be determined by the following:

Pitch diameter, $D_p = P \times D_{pf}$

Root diameter (max.), $D_r = 2 \times (\text{C.L. to Flat})$

$$\text{C.L. to Flat} = \sqrt{\left(\frac{D_p}{2}\right)^2 - \left(\frac{P}{2}\right)^2} - \frac{1}{2} F$$

Chain clearance circle (max.), $C_c = 2 \times (\text{C.L. to Flat}) - \frac{1}{4} \text{ in. (6.4 mm)}$

Outside diameter, $D_o = D_p \times K$ (see Table 6)

Total pitch line clearance, $C_p = D$

Pitch diameter factor, $D_{pf} =$ (see Table 6)

Topping radius, $R_t = 0.63 \times P$

Pressure angle, $\theta =$ (see Table 6)

Pocket radius (max.), $R_p = \frac{F}{2}$

Total width (max.), $W_t = 0.95 \times Z$

Side slope, $S_s = \text{approx. } 0.12 \times W_t$ [not to exceed 0.38 in. (9.6 mm)]

Nomenclature:

D = pin diameter (see Tables 2, 3, 4)

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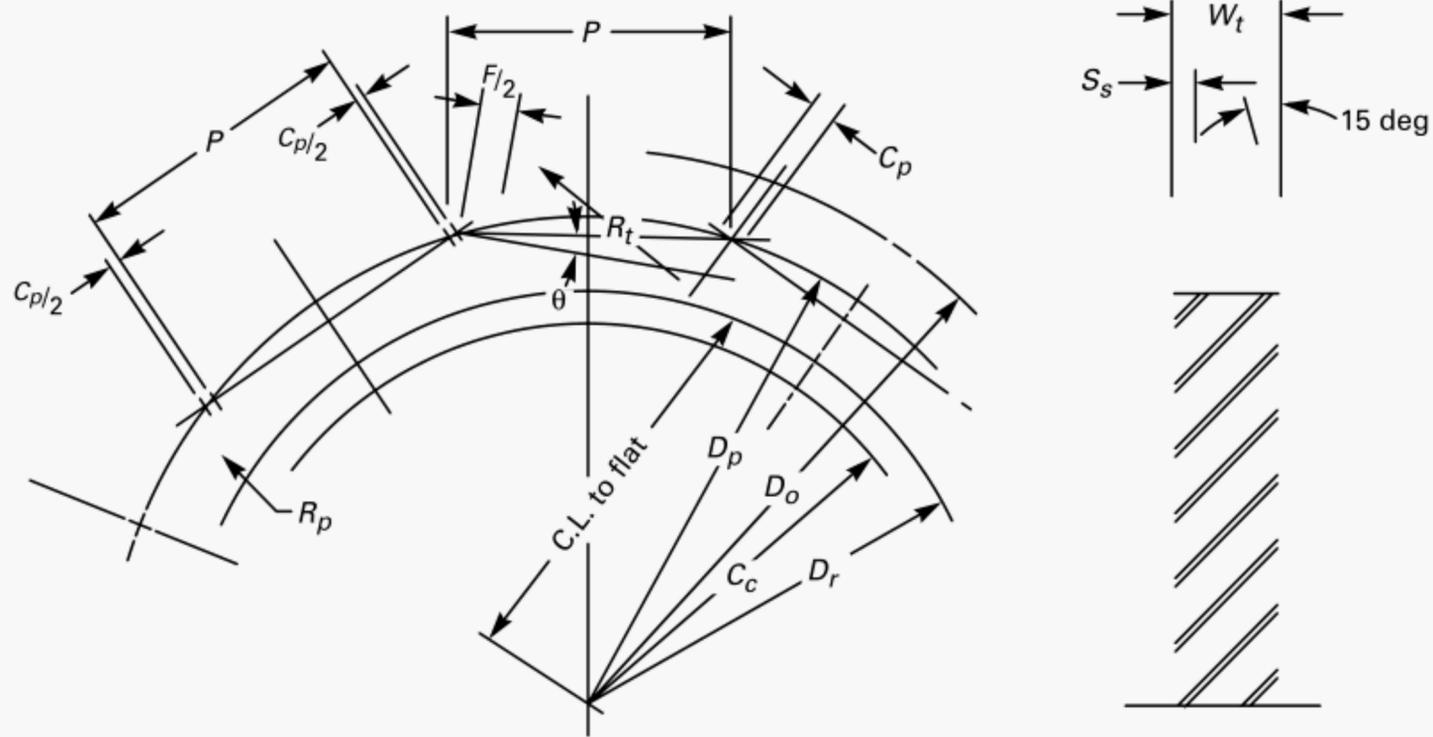
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FIG. 6 SPROCKET TOOTH FORM A



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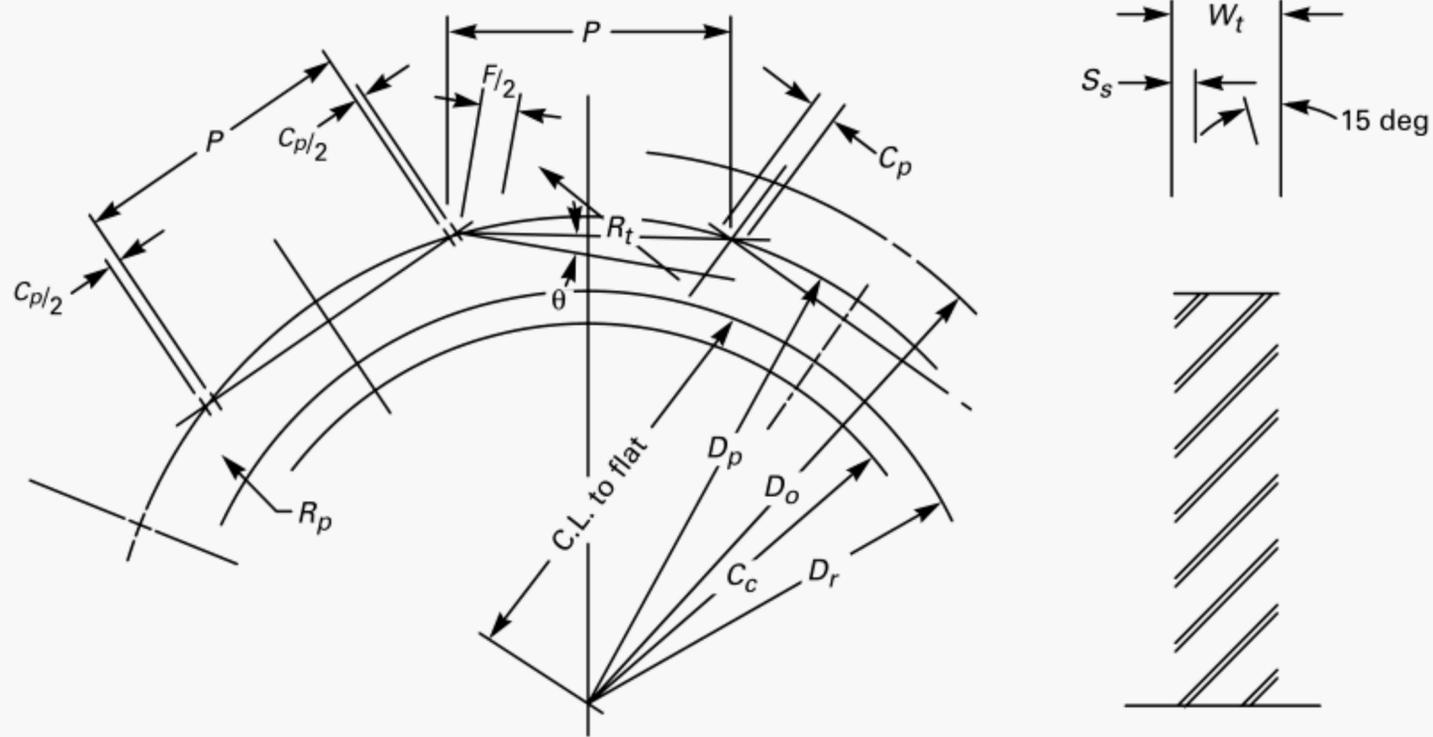
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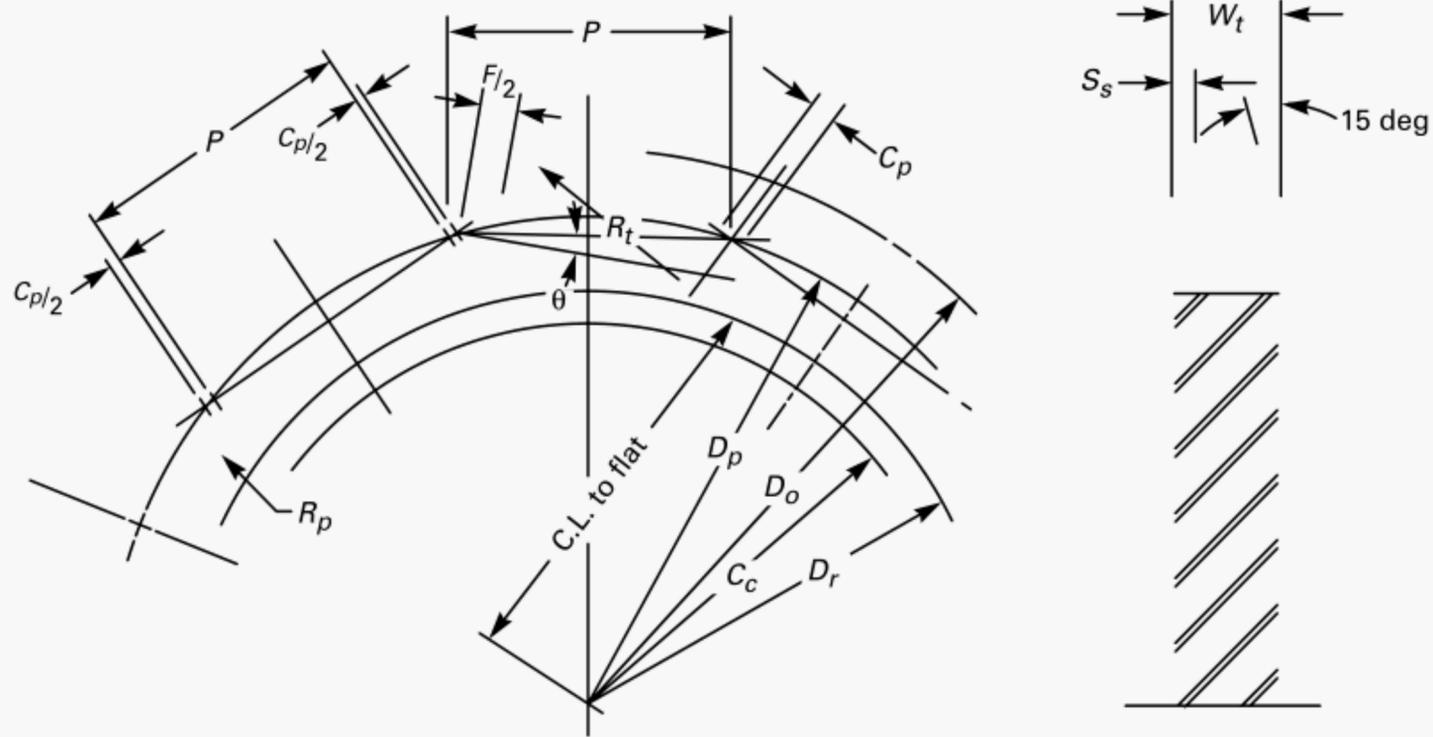
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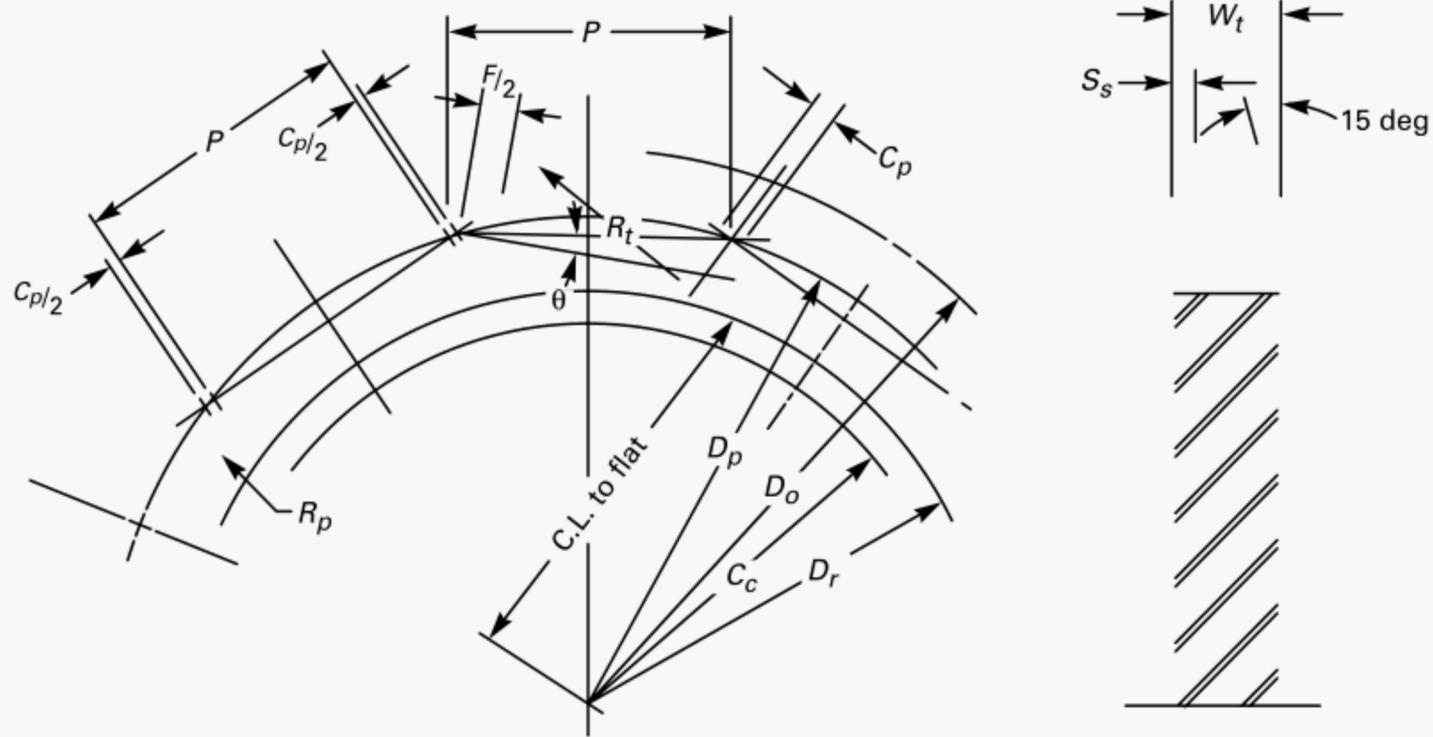
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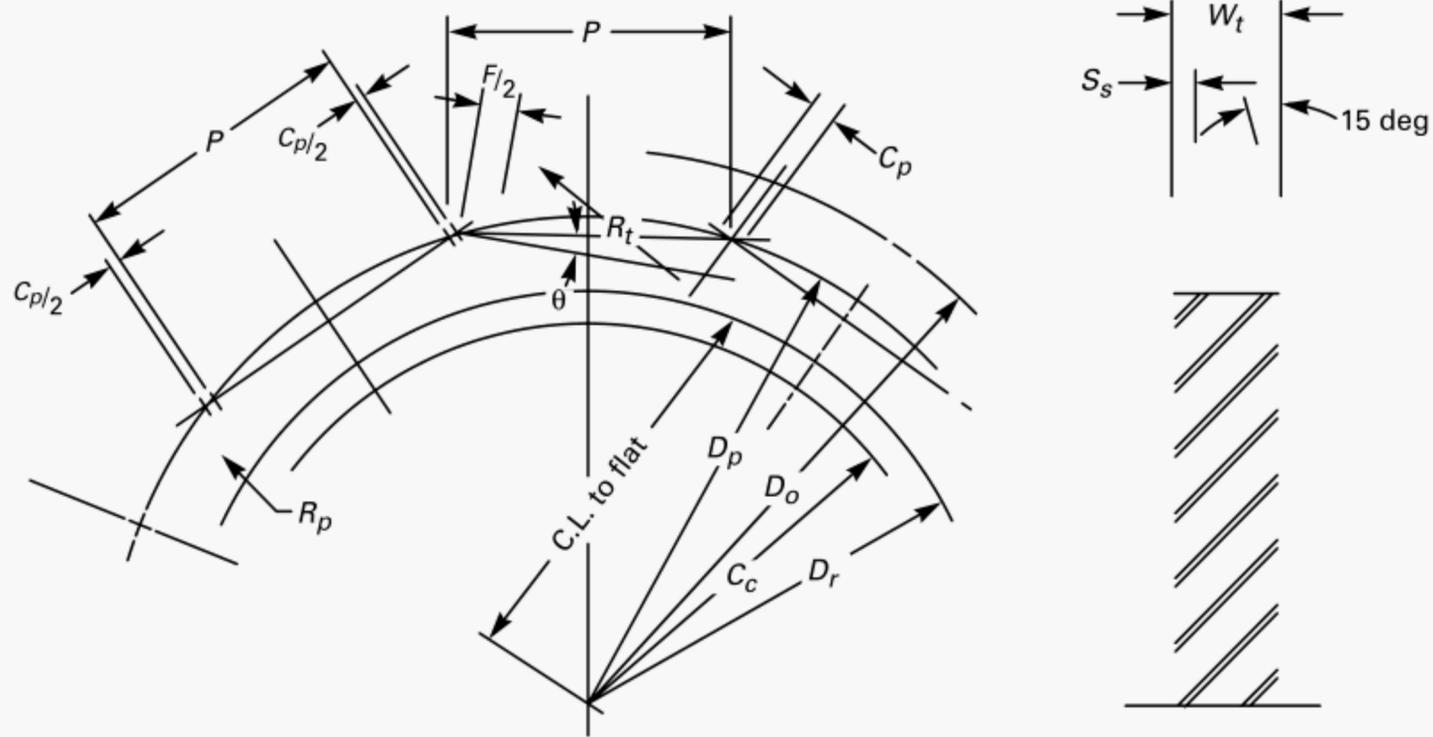
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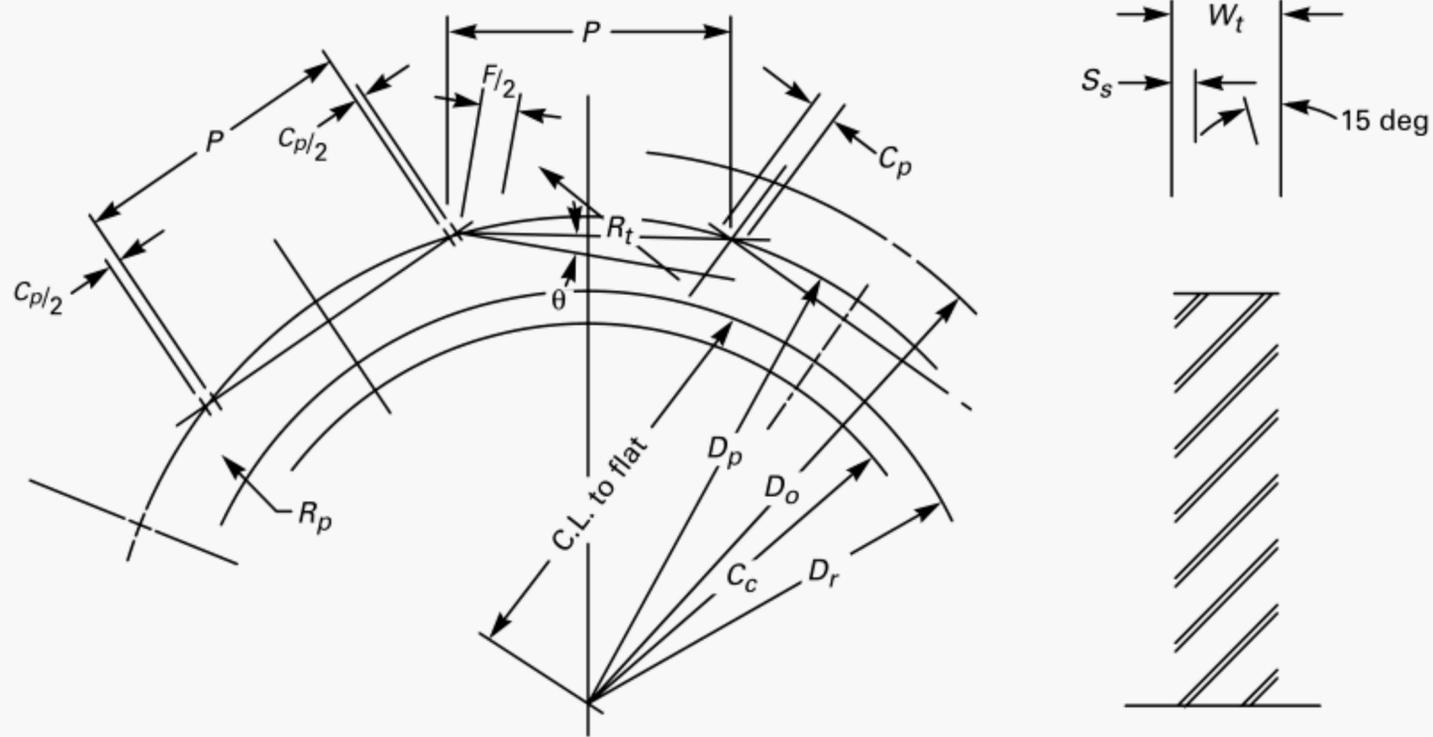
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